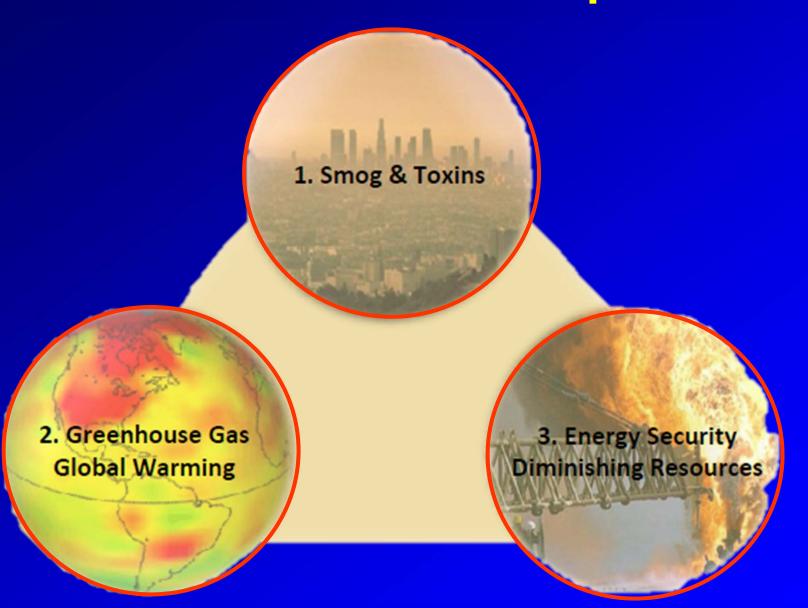
Experimental investigation of ethanol-gasoline dual-fuel on particle emissions at the exhaust of a small displacement engine

F. Catapano, S. Di Iorio, P. Sementa, B. M. Vaglieco



Istituto Motori CNR, Naples Italy

Main concerns on transportation



Introduction





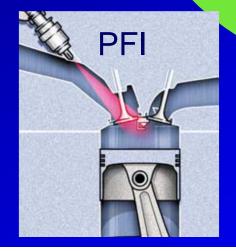
Fuel economy





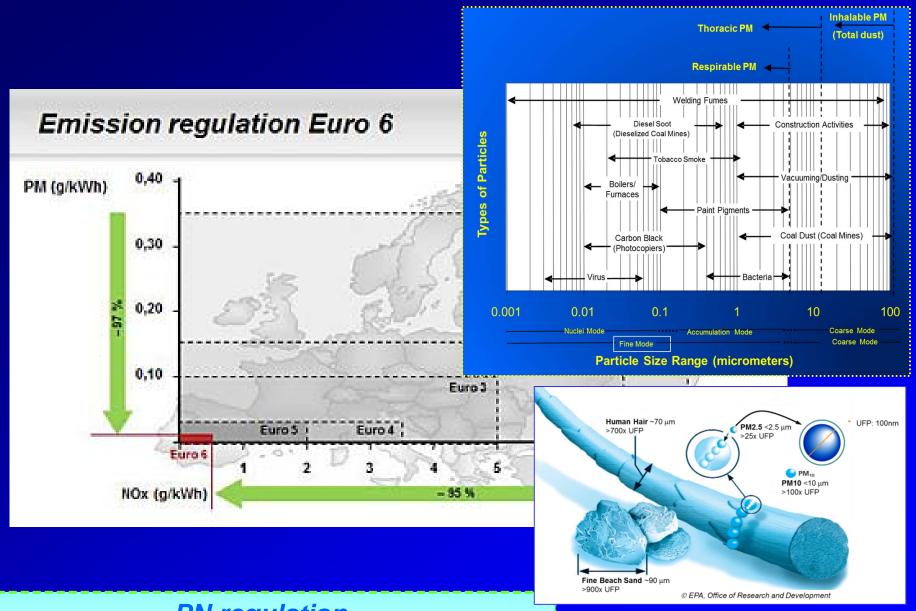


Engine performance



Increase of PM emissions

Time for mixture preparations charge heterogeneity
Time for fuel evaporation fuel-croplets
Wall impingement



PN regulation

6.0×10¹² 1/km of particles smaller than 23 nm

Introduction





Engine performance



Lowest pollutant emissions



Carbon footprint

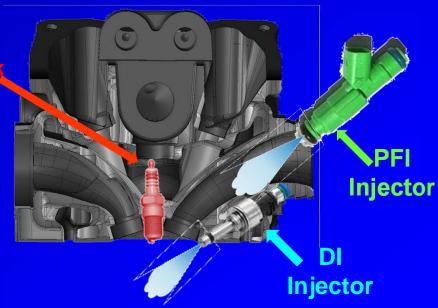


Octane enhancer Oxygenated fuel

Objective

Characterization of particle emissions from ethanol/gasoline dual fueling engine



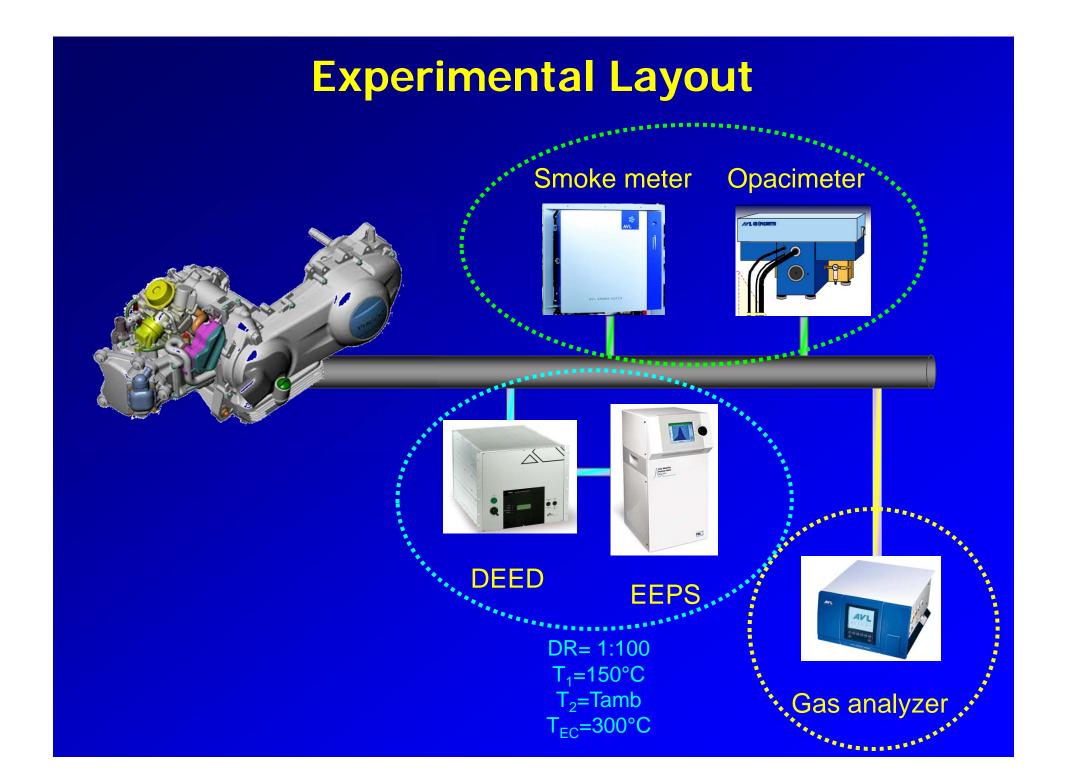


SI Engine

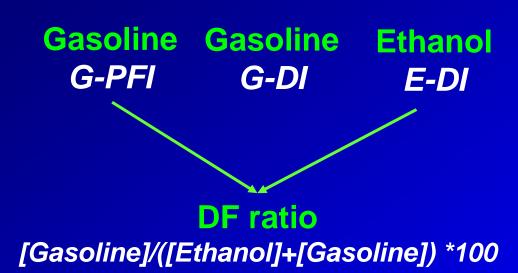




Engine type	4-stroke single cylinder	
Displacement (cm ³)	250	
Bore (mm)	72	
Stroke (mm)	60	
Engine size (cm³)	522.1	
Bowl Volume (cm ³)	19.7	
Maximum Torque [Nm]	20 Nm @ 5500 rpm	
Maximum Power [kW]	16 kW @ 8000 rpm	
Compression ratio	10.5:1	
Injector Type	PFI	DI
	Commercial	Prototypal
Number of Nozzle Holes	3	6



Engine operating conditions



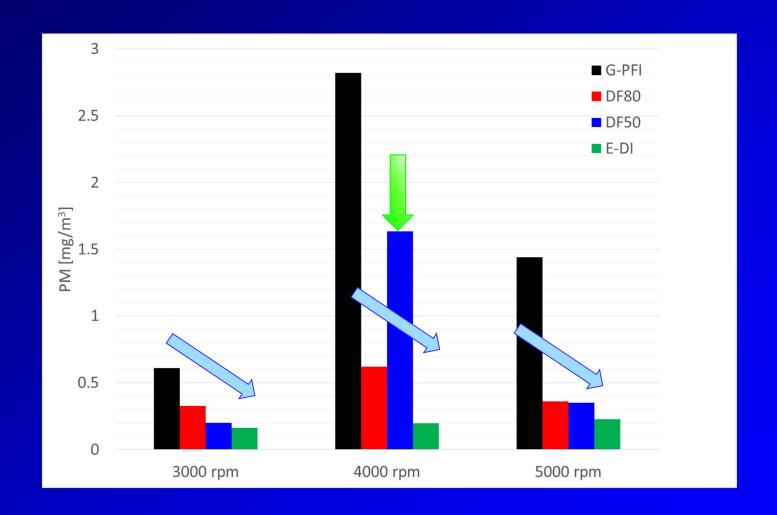
DF80
DF50

Urban
driving conditions

3000 rpm WOT
4000 rpm WOT
5000 rpm WOT

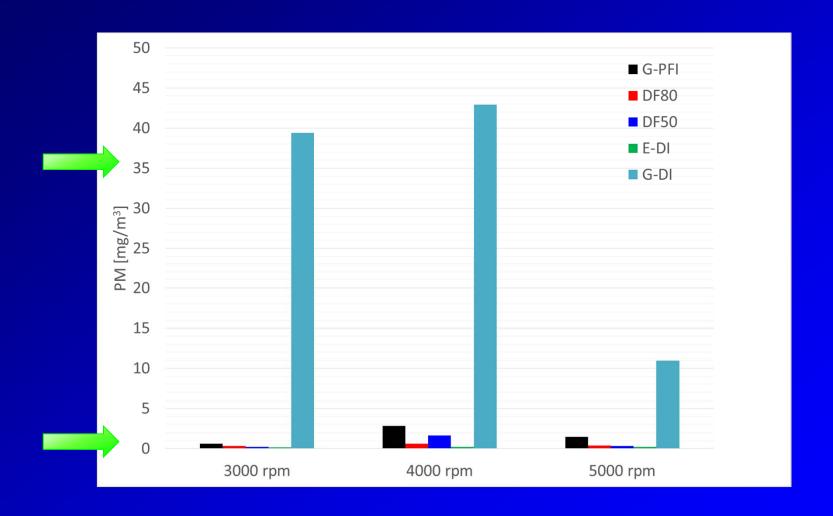
SOI 230 PFI
315 DI

Particle Mass Concentration



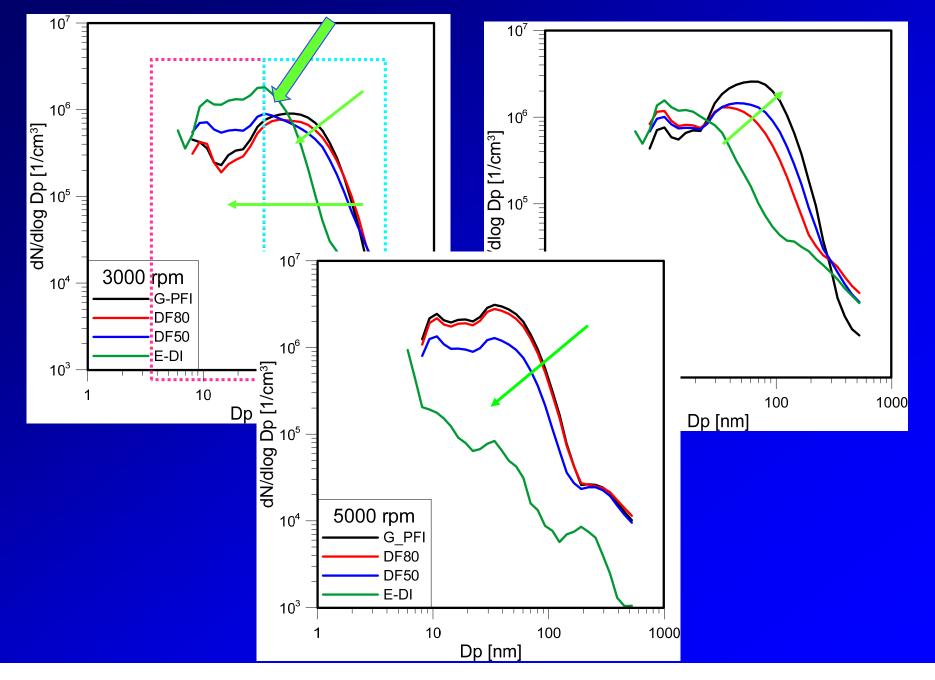
DF80: 80% gasoline (PFI) 20% ethanol (DI) DF50: 50% gasoline (PFI) 50% ethanol (DI)

Particle Mass Concentration



DF80: 80% gasoline (PFI) 20% ethanol (DI) DF50: 50% gasoline (PFI) 50% ethanol (DI)

Particle Size Distribution



Particle Size Distribution

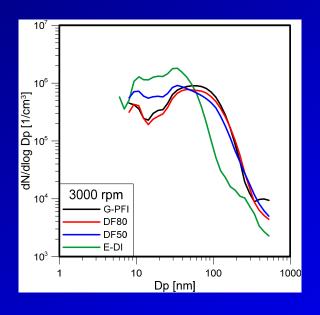
Oxygen content

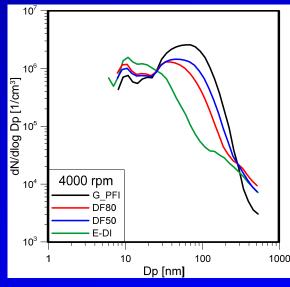
Reduced soot precursor formation Enhanced soot oxidation Volatility properties

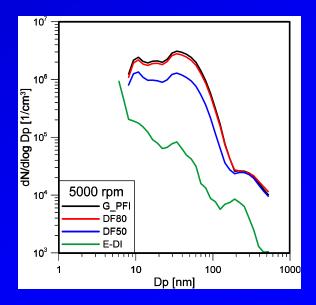
Evaporation conditions



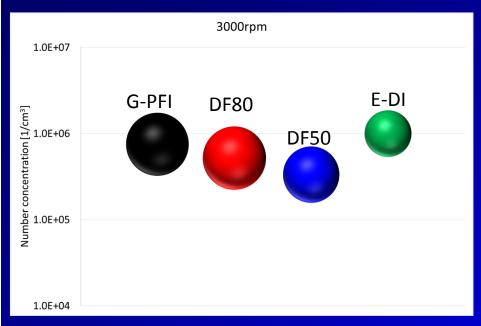
Enhanced evaporation of lighter compounds of gasoline: residual fuel will mainly contain heavier and highly sooting hydrocarbons

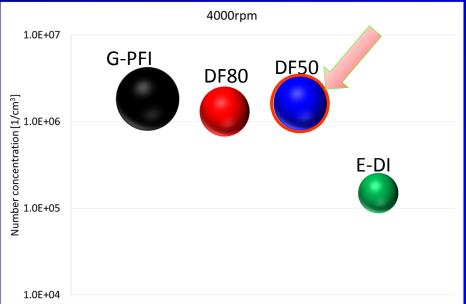






Particle Size and Number



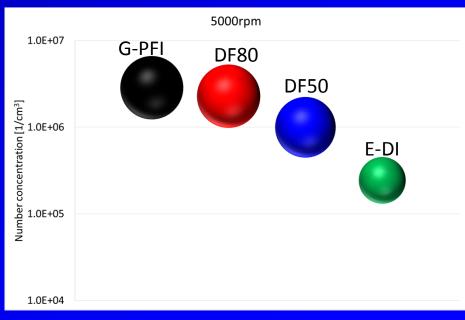


Particle Number

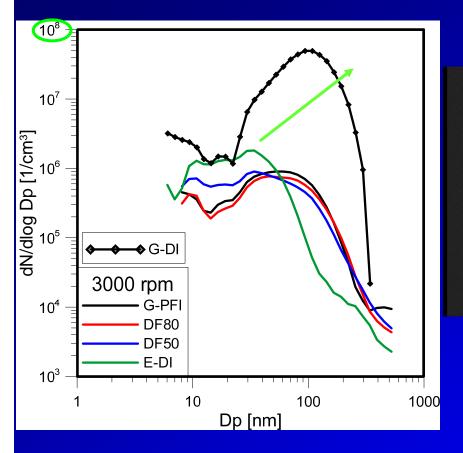
$$N_i = \frac{dN_i / d \log D_p}{16}$$

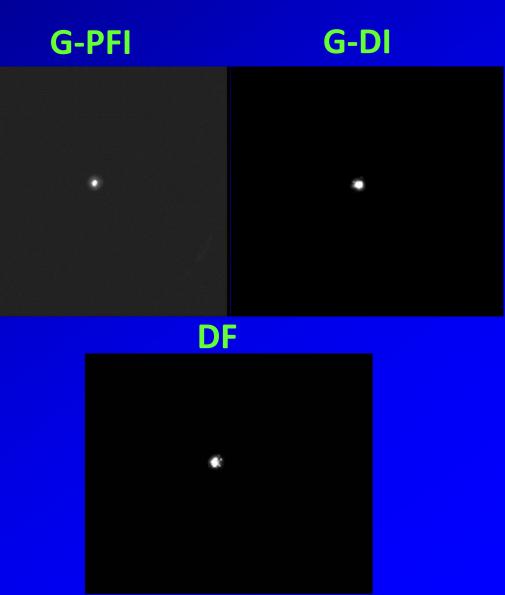
Mean Particle Diameter

$$D_m = \frac{\sum_{i=1}^n D_{p,i} \cdot N_i}{\sum_{i=1}^n N_i}$$

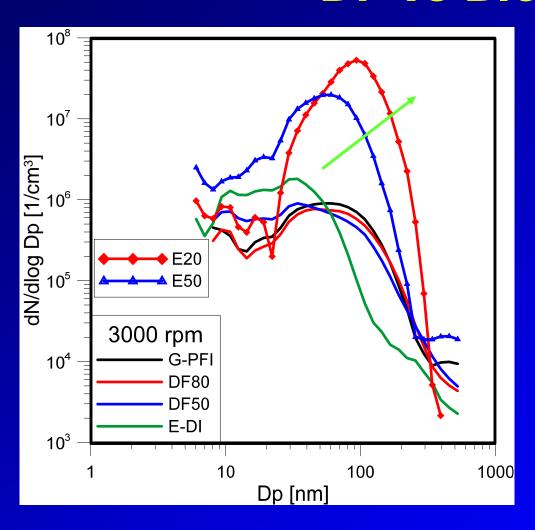


Particle Emissions: DF vs GDI





Particle Emissions: DF vs Blends



DF80

80% gasoline (PFI) 20% ethanol (DI)

E20

80% gasoline (DI) 20% ethanol (DI)

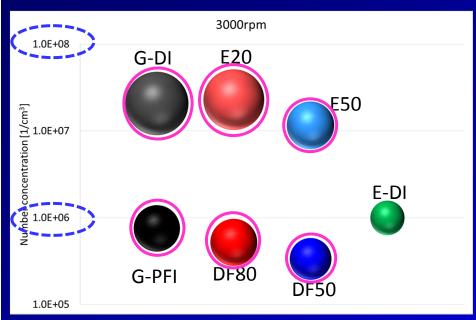
DF50

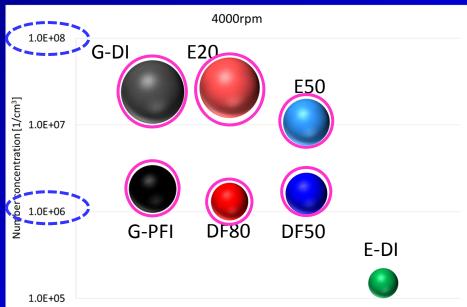
50% gasoline (PFI) 50% ethanol (DI)

E50

50% gasoline (DI) 50% ethanol (DI)

Exhaust Particle Size and Number





DF80:

80% gasoline (PFI) 20% ethanol (DI)

E20:

80% gasoline (DI) 20% ethanol (DI)

DF50:

50% gasoline (PFI) 50% ethanol (DI)

E50:

50% gasoline (DI) 50% ethanol (DI)

Conclusions 1/2

The effects of the **ethanol/gasoline dual fueling** on the particle emissions in a small **SI** engine was investigated.

Engine worked at **3000 rpm**, **4000 rpm** and **5000 rpm full load** representative of urban driving conditions.

Engine was fueled with pure ethanol and gasoline and different dual fuel ratio of ethanol in gasoline (*DF80-DF50*).

A smoke meter were used for particle concentration measurement.

A Engine Exhaust Particle Sizer (**EEPS**) was used for counting and sizing of the particles in the size range **5.6-560 nm**.

Conclusions 2/2

Particle mass decreases with ethanol content except for DF50 @ 4000 rpm;

Particle number & size decreases with ethanol content except for DF50 @ 4000 rpm.

The particle emissions is strongly affected by:

- > Fuel:
- <u>oxygen content</u>: sooting reduction tendency;
- <u>evaporation rate</u>: selective and enhanced evaporation of ethanol and lighter compounds of gasoline: favorable conditions for the formation of particles from the sooting compounds of gasoline;
- > Engine operating point:
 - <u>In-cylinder temperature:</u> affects evaporation conditions.







Thank you for the attention



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