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Asymmetric and Variable Cell Geometry Diesel Particulate Filters

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Outline



Motivation

Theoretical Analysis

Pressure drop and soot deposit growth

Validation with Experiments

Conclusions

Emission Control: A Chemical Plant in the Exhaust



•Adding devices = more pressure drop, more space in system layout

• Increasing catalyst loads = more pressure drop, less space in device

✓ Optimization requires overcoming these constraints

✓ Focus on cell geometry and layout

Evolution of DPF Cell Geometries



ASYMMETRIC DESIGNS



Young et al SAE 2004-01-0948



dieselnet.com

WAVY



Bardon et al SAE 2004-01-0950

NEW DESIGNS?



Flow Re-adjustment in Asymmetric Cells

OS WAVY ACT computational computational domain domain computational domain inlet channel inlet channel inlet channel wall wall clean loaded clean loaded clean loaded wall wall wall wall wall wall soot soot cake cake søot cake

Konstandopoulos et al, SAE 2005-01-0946

- Flow continuously readjusts according to the wall resistances
- Soot deposits form on all walls

Flow Re-adjustment in Asymmetric Cells



- Follow up studies (e.g. Wurzenberger et al SAE 2007-01-1137, Tang et al SAE 2009-01-127, Aravelli et al 2007-01-0920) have addressed modeling aspects of asymmetric cell DPFs, with varying degrees of simplification.
- In all cases a single valued "wall/filtration velocity" is adopted without explicit considerations of additional flow paths over the cell geometry.

Cell Geometries Studied



General model for VPL which can be reduced to one for OS

Model Formulation: Two Types of Inlet Channels-1

Exhaust gas mass balances for each channel (1, 2, 3)

$$\begin{vmatrix} A_1 \frac{\partial u_1}{\partial z} = -\Pi_1 v_{w12} - \Pi_1 v_{w13} - \Pi_2 v_{w2} \end{vmatrix}$$

$$\mathbf{A}_2 \frac{\partial \mathbf{u}_2}{\partial \mathbf{z}} = \boldsymbol{\Pi}_1 \mathbf{v}_{w12} - \boldsymbol{\Pi}_1 \mathbf{v}_{w23} - \boldsymbol{\Pi}_3 \mathbf{v}_{w3}$$

$$A_{3} \frac{\partial u_{3}}{\partial z} = \Pi_{1} v_{w13} + \Pi_{2} v_{w2} + \Pi_{1} v_{w23} + \Pi_{3} v_{w3}$$

where u_1, u_2, u_3 are the cross section averaged velocities

Axial Momentum Balances for each channel (1, 2, 3)

$$\frac{\partial P_1}{\partial z} = -\alpha_1(t)\mu u_1 \frac{A_1}{A_{1t}(t)}$$
$$\frac{\partial P_2}{\partial z} = -\alpha_2(t)\mu u_2 \frac{A_2}{A_{2t}(t)}$$

 $= -\alpha_{3}\mu u_{3}$

 Π_{i} : Perimeter of each channel, i

 \boldsymbol{A}_i : Cross section of each channel, $\boldsymbol{A}_{it}(t)$: evolving area of cell

 α_i (t): Evolving friction coefficient of each cell

5 flow paths/velocities



Model Formulation: Two Types of Inlet Channels-2

Wall Momentum Balances (Pressure Drop through each flow path)

$$\mathbf{v}_{w12} = (\frac{\mu w_{s1} k_w}{k_w} + \frac{\mu (w_{12} + w_{21})}{k_s})^{-1} (\mathbf{P}_1 - \mathbf{P}_2)$$

$$\mathbf{v}_{w13} = (\frac{\mu w_{eff}}{k_{sw}} + \frac{\mu w_{12}}{k_s})^{-1} (\mathbf{P}_1 - \mathbf{P}_3)$$

$$v_{w2} = (\frac{\mu w_{s2}}{k_w} + \frac{\mu w_1}{k_s})^{-1}(P_1 - P_3)$$

$$v_{w3} = (\frac{\mu w_{s3}}{k_w} + \frac{\mu w_2}{k_s})^{-1} (P_2 - P_3)$$

$$v_{w23} = \left(\frac{\mu w_{eff}}{k_w} + \frac{\mu w_{21}}{k_s}\right)^{-1} (P_2 - P_3)$$



 $w_{\text{eff}}\!\!:$ is determined by a separate flow problem over the wall cross-section

Model Formulation: Two Types of Inlet Channels-3

Soot Deposits Evolution





 $\Omega(x)=x$ for x>0 and $\Omega(x)=0$ for x ≤ 0 .



Soot Transport and Deposition

$$A_1 u_1 \frac{\partial \varphi_1}{\partial z} = -\varphi_1 \Pi_1 \Omega (-(v_{w12} + v_{w13}))$$

$$A_2 u_2 \frac{\partial \varphi_2}{\partial z} = -\varphi_2 \Pi_2 U(v_{w12} - v_{w23})$$

 ϕ 1, ϕ 2 : the local soot mass fraction in gas phase

Cell cross-section evolution

$$A_{1t}(t) = A_1 - \Pi_1 W_1 - \Pi_2 W_2$$

$$A_{2t}(t) = A_2 - \Pi_1 W_{r1} - \Pi_3 W_3$$

 $\begin{array}{l} \textbf{Boundary conditions} \\ P_3 = P_{atm} \text{ at } z = L, \\ A_1 u_1 + A_2 u_2 = (A_1 + A_2) u_o, \\ \phi_1 = \phi_2 = \phi_{in} \text{ at } z = 0 \end{array}$

where u_o is the average inlet velocity ϕ_{in} is the inlet soot mass fraction.

Solution procedure

- Outer initial value problem (evolution equations of the deposits) explicit intefration
- Inner boundary value problem (flow and soot transport) solved at each step of outer problem, fulfilling BC with Newton-Raphson
- Advance soot deposit thickness and evolve cross section



Samples Used for Validation – 4 OS DPFs

FILTER TYPE	OS	OS1	OS2	OS3	OS4
Diameter	mm	143.8	143.8	143.8	143.8
Length	mm	150.0	150.0	150.0	150.0
Wall permeability (x1E12)	m²	1.00	1.00	1.00	3.00
Oct-Sq wall thickness, w1	mm	0.4	0.4	0.4	0.4
Oct-Oct wall thickness, w2	mm	0.4	0.4	0.4	0.4
Oct side, C _{w1}	mm	1.77	1.96	2.17	2.44
Sq side, C _{w2}	mm	1.42	1.23	1.02	0.75



Soot loading of OS samples: Experiment and Model

Deep bed calculation not-included to see intrinsic curvature effects of Pressure Drop curve



Evolution of Wall Mass-Flux Ratio: (O to O /O to S)



Samples Used for Validation – 4 VPL DPFs

VPL	A	В	С	D
mm	143.8	143.8	143.8	143.8
mm	108.0	140.5	193.2	193.2
m²	4.00	4.00	3.203	3.203
mm	0.254	0.254	0.254	0.176
mm	0.359	0.359	0.359	0.251
mm	1.497	1.497	1.497	1.575
mm	0.962	0.962	0.962	1.039
	VPL mm mm mm mm mm	VPL A mm 143.8 mm 108.0 m² 4.00 mm 0.254 mm 0.359 mm 1.497 mm 0.962	VPLABmm143.8143.8mm108.0140.5m²4.004.00mm0.2540.254mm0.3590.359mm1.4971.497mm0.9620.962	VPLABCmm143.8143.8143.8mm108.0140.5193.2m²4.004.003.203mm0.2540.2540.254mm0.3590.3590.359mm1.4971.4971.497mm0.9620.9620.962



Soot loading of VPL samples: Experiment and Model



Soot deposits of VPL samples: Experiment and Model



Evolution of filtration velocities in VPL DPF



Optimization of DPF designs for constant volume



Conclusions



- Asymmetric and Variable Cell geometry DPFs introduce many complexities into the standard simulation framework of DPFs by requiring the explicit treatment of additional flow paths in order to properly capture the flow dynamics through the structure.
- Relevant wall fluxes and velocities have been identified and simulated. Their evolution at long times (high values of soot loafing) indicates that in the case of OS design a constant wall flux ratio is established and can be used as a metric to select DPFs with lower pressure drops. In the more complex case of the VPL design a clustering of all but one filtration velocity towards a common value is observed as a result of the complex interactions among the different flow paths.
- Pressure drop in the OS and VPL DPFs still follows a linear evolution with respect to soot load as the different flows through the common (inlet-inlet) and conventional (inlet-outlet) flow paths readjust to transport and deposit the soot particles through the path of least resistance.
- The advent of AVC DPF designs with many degrees of freedom with respect to filtration/wall velocities, leads to DPF systems with substantially reduced pressure drop compared to the state-of-the-art.

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WORKSHOP ANNOUNCEMENT- Oct. 16-17, 2014

אttp://aptstep.certh.gr Confire WORKSHOP ו Combustion Emission Control for Clean and Efficient Vehicles October 16-17, 2014 Thessaloniki, Greece

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Future Fuels - Future Engines Multifunctional Emission Control Devices Challenges in Low Emissions Measurement & Testing Air Quality Issues and Retrofit Approaches

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Thank you for your attention!

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